



Development and Validation of a Test Procedure for Determining the System Power of Hybrid Electric Vehicles

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Importance of vehicle power rating

- Passenger vehicles are commonly assigned a power rating
 - Allows comparison of performance between different vehicles
 - Other uses: certification, classification, taxation, marketing

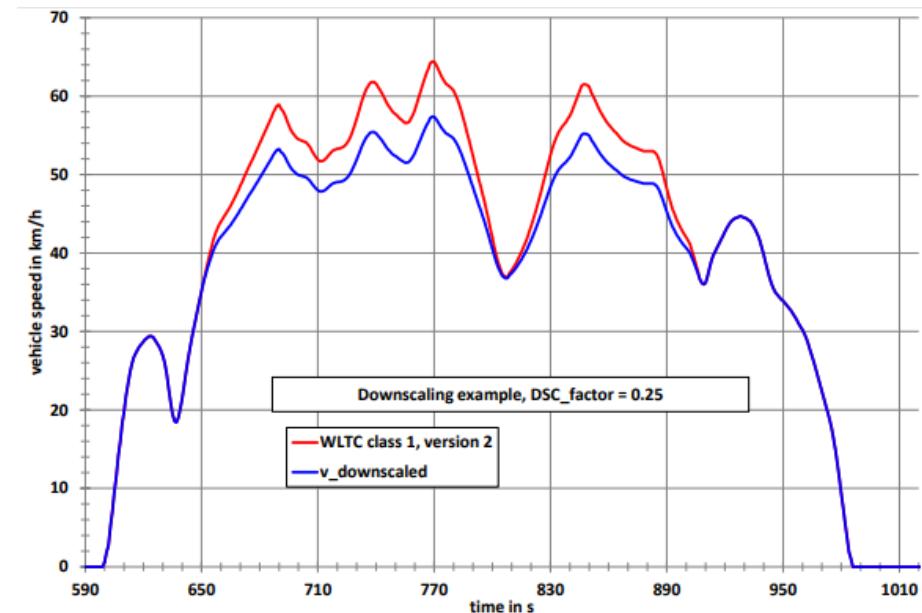


2.8%



Relation to WLTP

- WLTP requires a vehicle power rating for classification and downscaling
 - Power-to-mass ratio is used for vehicle classification
 - Power rating is used to downscale the drive trace for low-powered vehicles
- Currently, HEVs are placed into Class 3 because there is no specified method for determining their power rating





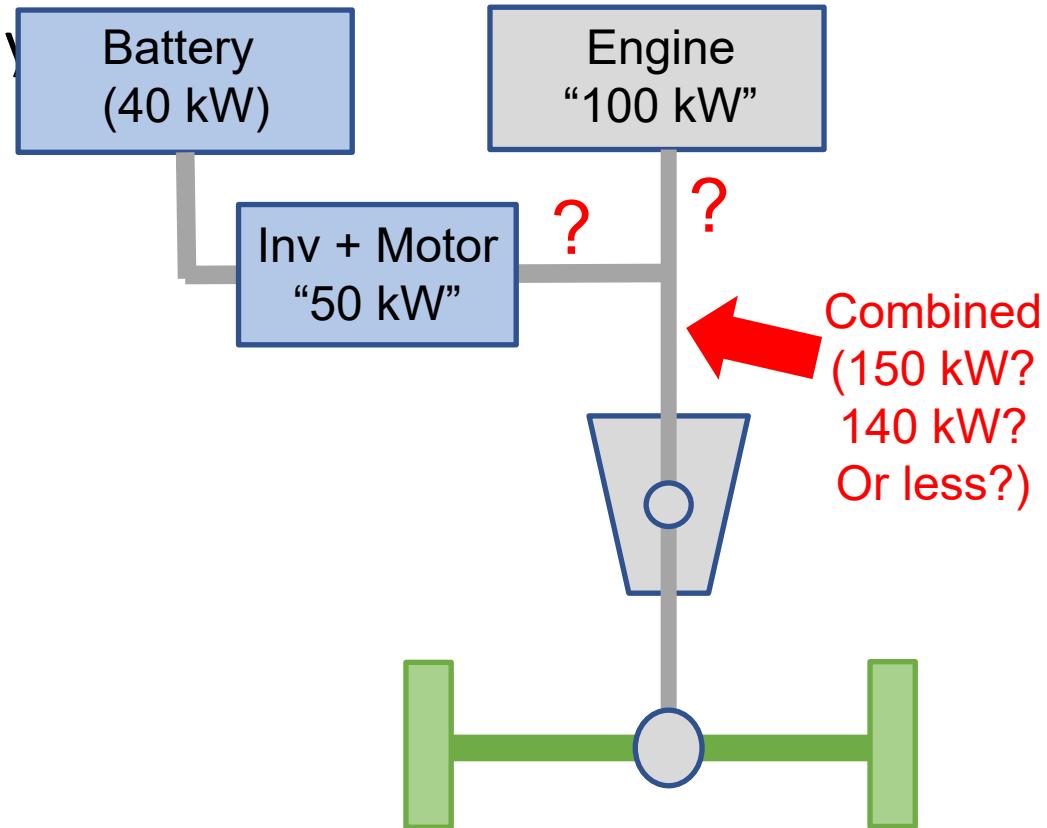
UN ECE mandate

- UN ECE is developing a Global Technical Regulation, “Determination of Electrified Vehicle Power (DEVP)” to provide for HEV power rating
- Delegated to Informal Working Group (IWG) on Electrified Vehicles and the Environment (EVE)
- Power determination test procedure will apply to light-duty:
 - HEVs
 - PHEVs
 - BEVs with more than one propulsion motor



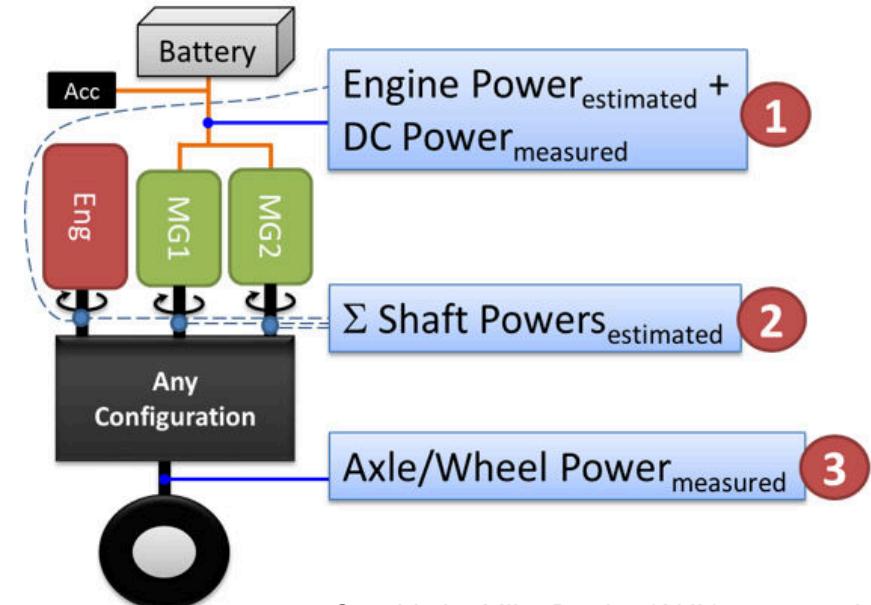
The difficulty with HEVs

- Rated power of a conventional vehicle is simply the rated engine power.
- HEVs combine more than one power source
 - It depends on how they are combined
 - Rated power might not be achieved
 - Rated power may be limited by battery
- So it cannot be a sum of the rated powers
- But it could be a sum of the actual power produced by each component.
- But how do we measure it?



Power measurement standards

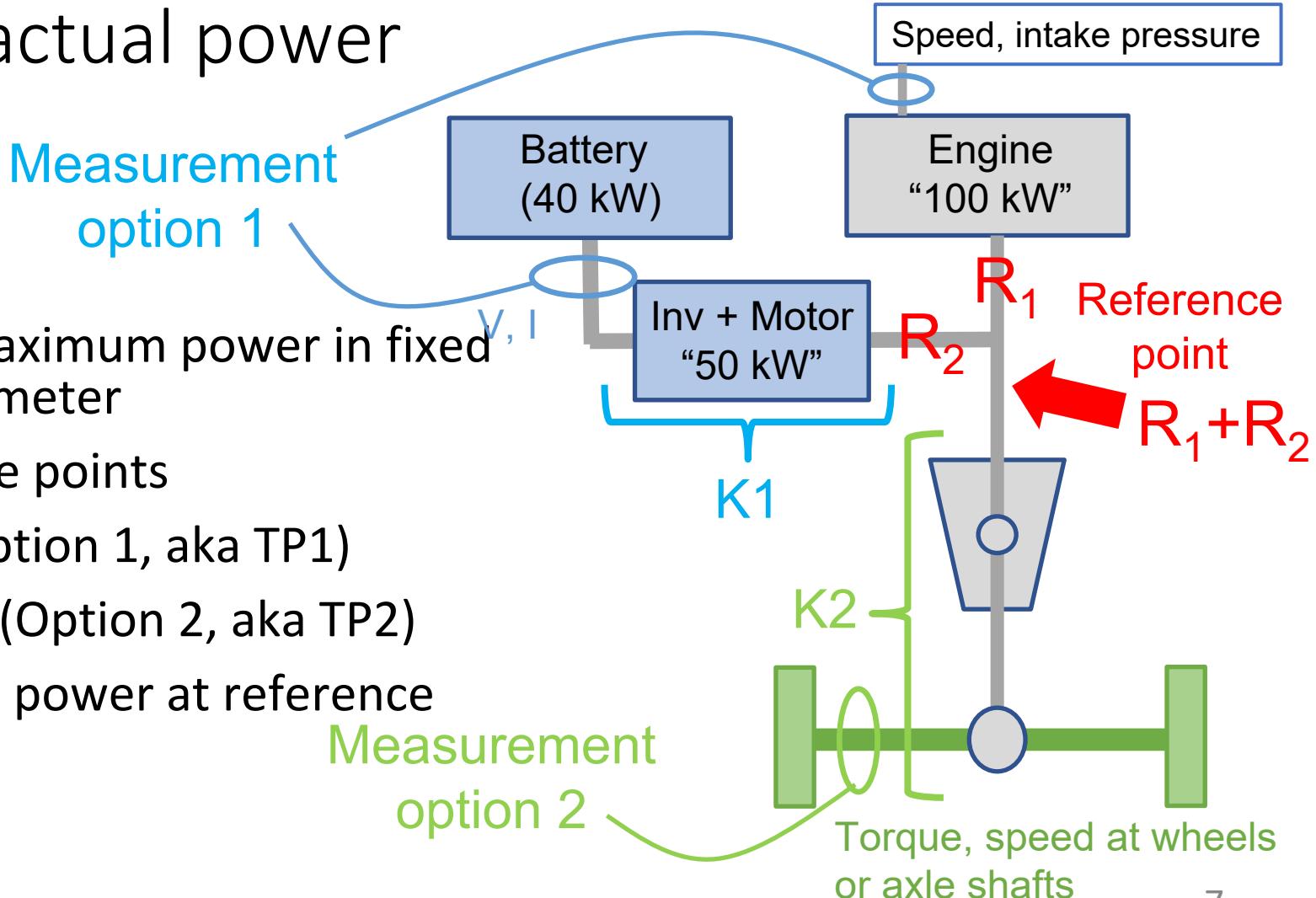
- Several standards bodies have developed solutions
 - SAE J2908 Vehicle Power Test for Electrified Powertrains
 - ISO 20762 Electrically propelled road vehicles -- Determination of power for propulsion of hybrid electric vehicle
 - Korea Automobile Testing & Research Institute (KATRI) has explored similar methods
- EVE IWG considered each of these
- Ultimately, we selected ISO 20762 as a basis
- The goal is to estimate actual shaft output power



Graphic by Mike Duoba (ANL) presented at EVE-16, Ottawa, October 2015

Approach: measuring actual power

- 1) Operate the vehicle at its maximum power in fixed speed mode of the dynamometer
- 2) Measure power at accessible points
 - Upstream of reference (Option 1, aka TP1)
 - Downstream of reference (Option 2, aka TP2)
- 3) Apply K_1 or K_2 to convert to power at reference point ($R_1 + R_2$)





Outline of procedure

- Vehicle soak and charge (if applicable)
- Conditioning cycle – 20 minutes at 60 kph
- Battery SOC recovery
- Identify speed of maximum power
 - Maximum pedal command at various fixed dynamometer speeds
- Conduct five repetitions at speed of maximum power
- Compute TP1 or TP2 and compute average of last 4 observations



TP1

Default = 0.85

$$P_{HEV \text{ system}} [\text{kW}] = \text{ICE power} + \left(\frac{U_{battery} \times I_{battery}}{1000} - P_{DCDC} - P_{auxiliaries} \right) \times K1$$

K1 is efficiency of [inverter + motor] during vehicle maximum power

TP2

$$P_{HEV \text{ system}} [\text{kW}] = \frac{P_{wheels}}{K2}$$

Default = 0.91 to 0.98

K2 is efficiency of transmission/gearbox during vehicle max power

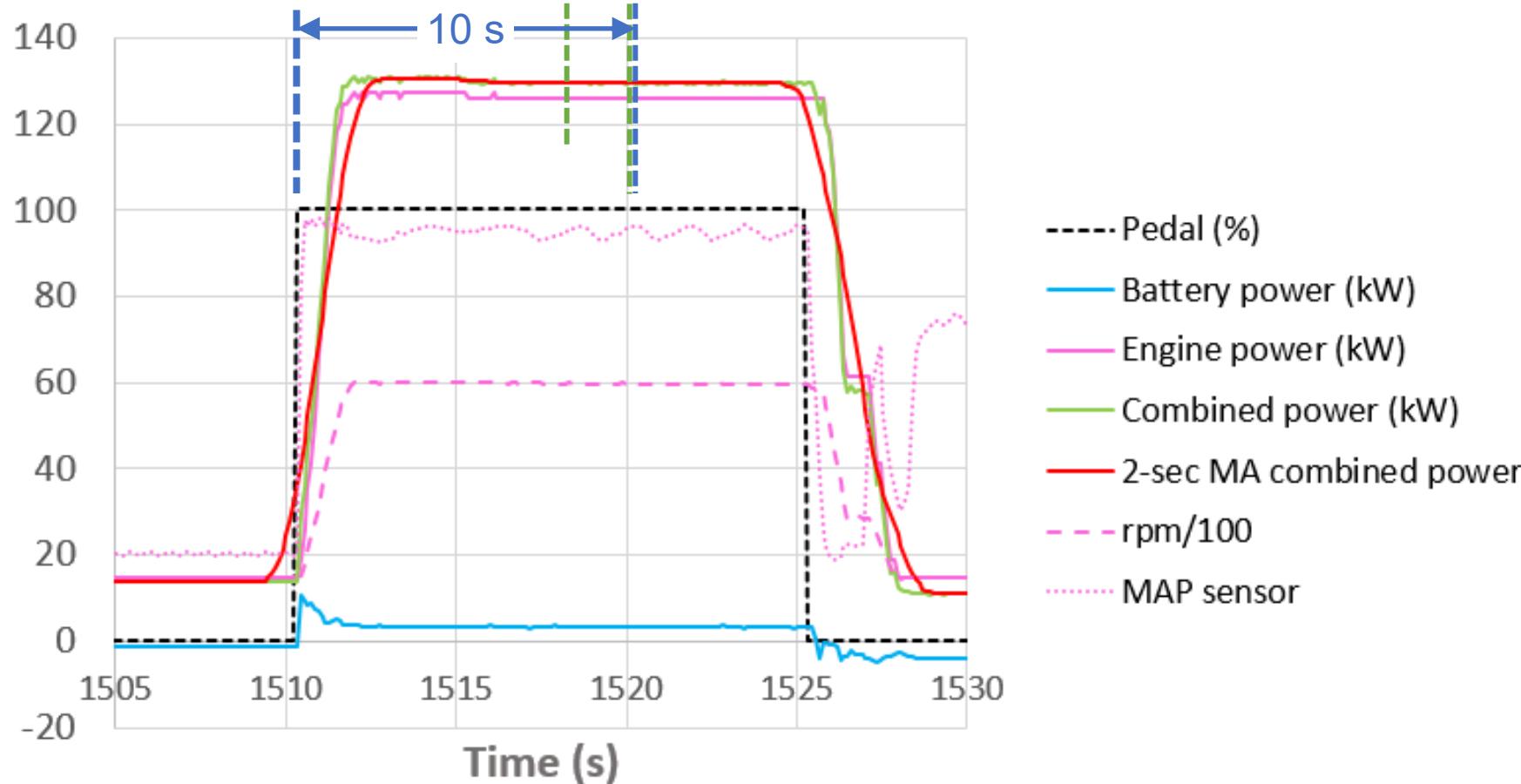


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Peak power = maximum of 2-sec moving average during 1st 10 seconds

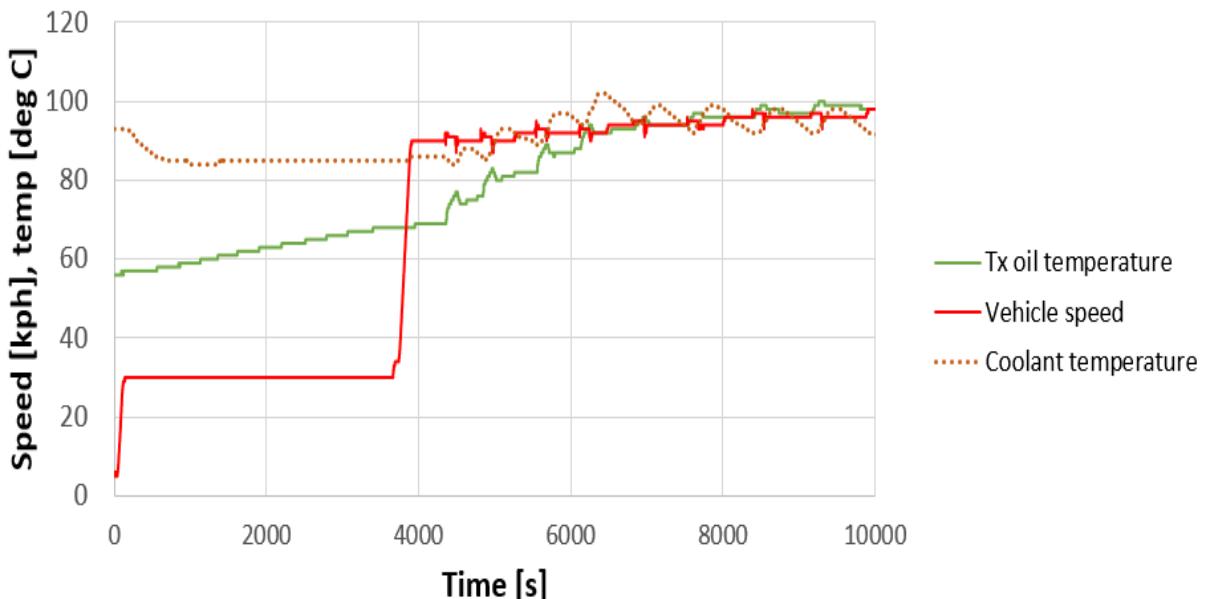
Sustained power = average power between 8th and 10th second





Results: Conditioning cycle

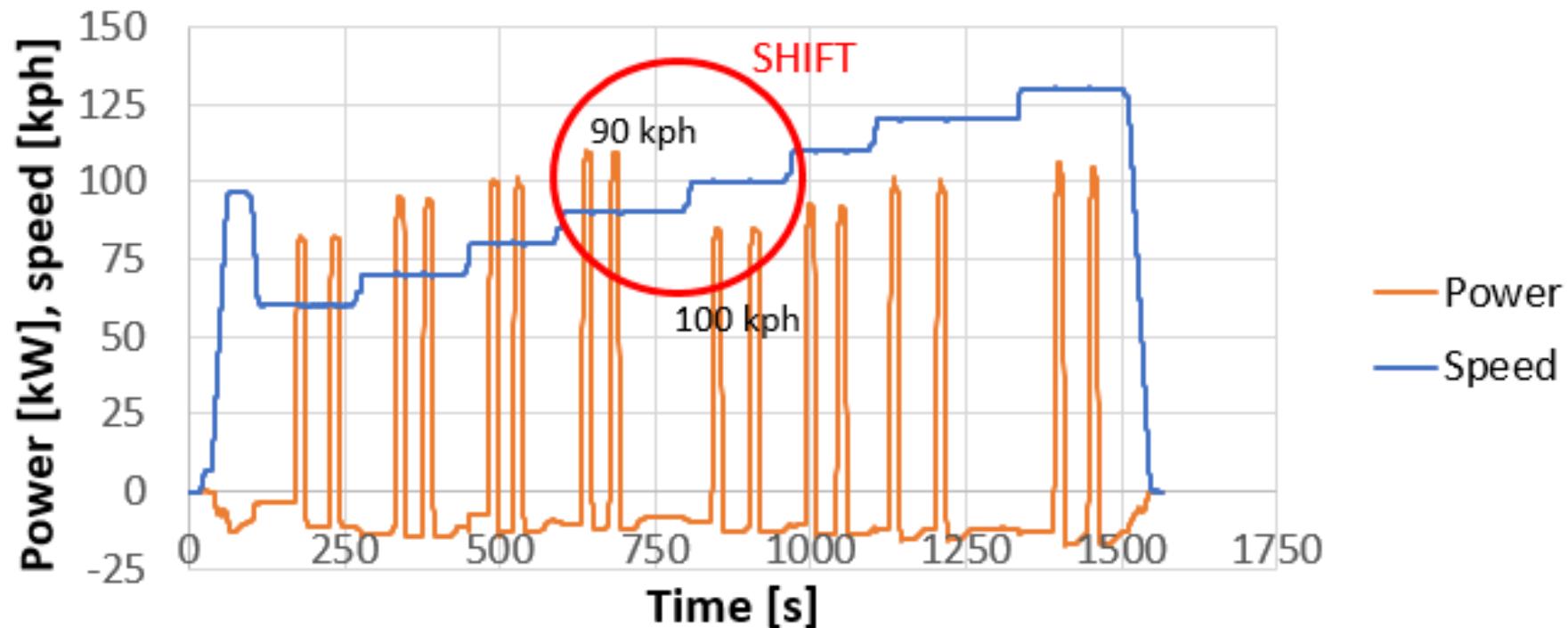
- Was adequate for most components to reach operating temperature
- Transmission oil temperature continued to rise
- Propose longer conditioning time, monitoring of temperature, or segmenting the test





Results: Identifying speed of maximum power

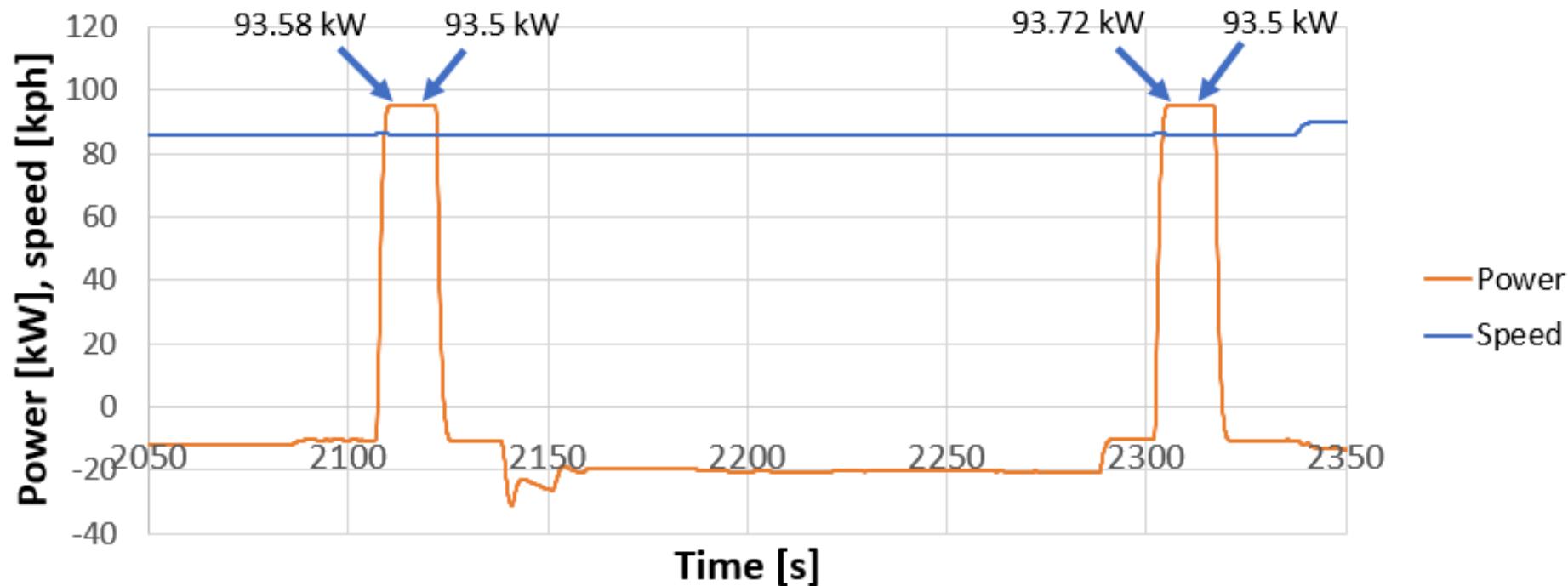
2013 Malibu Eco





Results: Identifying speed of maximum power

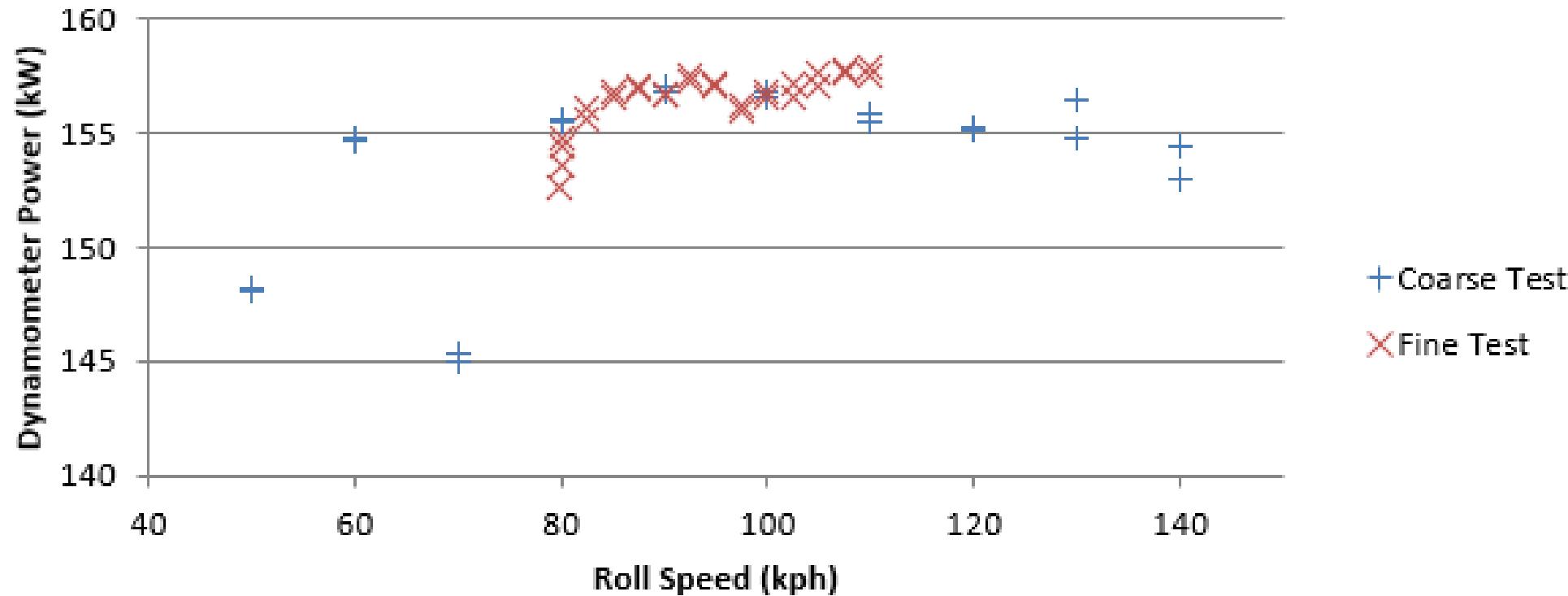
2013 Chevy Volt





Results: Identifying speed of maximum power

BMW 530e HEV





INTERNATIONAL ELECTRIC VEHICLE SYMPOSIUM & EXHIBITION



Results: Difference between TP1 and TP2

	Peak power			Sustained power		
	TP1	TP2	Difference	TP1	TP2	Difference
First pulse	130.73 kW	121.07 kW	-9.66 kW	129.57 kW	120.15 kW	-9.42 kW
Second pulse	130.04 kW	119.87 kW	-10.17 kW	129.54 kW	118.99 kW	-10.55 kW
Difference	-0.69 kW	-1.2 kW		-0.03 kW	-1.16 kW	

2013 Malibu Eco

	Peak power			Sustained power		
	TP1	TP2	Difference	TP1	TP2	Difference
First pulse	94.31 kW	103.29 kW	+8.98 kW	92.91 kW	101.00 kW	+8.09 kW
Second pulse	93.21 kW	102.31 kW	+9.10 kW	92.67 kW	101.19 kW	+8.51 kW
Difference	-1.10 kW	-0.98 kW		-0.24 kW	+0.18 kW	

2013 Chevy Volt



Downstream efficiency check

- No matter which TP, we know power at wheels (from dyno rolls)
- We can compare the computed TP value to the power at wheels
 - Suppose result = 100 kW, and 90 kW was measured at rolls
 - Implied downstream efficiency (IDE) = $90 / 100 = 90\%$

TP1 result was smaller than power measured at wheels

	Sustained		Difference
	TP1	TP2	
First pulse	87.0%	93.8%	+6.8
Second pulse	86.2%	93.8%	+7.6
Difference	-0.8	0.0	

Implied downstream efficiency, 2013 Malibu Eco

	Sustained		Difference
	TP1	TP2	
First pulse	103.7%	95.4%	-8.3
Second pulse	104.1%	95.4%	-8.7
Difference	+0.4	0.0	

Implied downstream efficiency, 2013 Chevy Volt



Role of default K factors

- Draft procedure provides default K factors as backup
 - Default K1 = 0.85 for all electric machines
 - Default K2 = 0.91 to 0.98 depending on gearbox type
- Result is sensitive to accuracy of the K factor with respect to the vehicle being tested
- If the respective K factors for TP1 and TP2 are not comparably accurate, TP1 and TP2 are likely to vary
- EVE IWG is considering elimination of default values in favor of verifiable manufacturer-supplied values



Accounting for tire losses in TP2

- Draft procedure allows collecting torque and speed from dyno rollers
- It does not specify how to account for tire losses (Crr) or slippage
- Losses can be estimated, but normal force is often unknown when vehicle is installed on the dyno
- Torque and speed meters, or hub dyno, will remove this uncertainty
- EVE IWG is planning to eliminate use of dyno rollers as an option



Areas for improvement

- A difference between TP1 and TP2 is undesirable
 - Use of default K factors contributes in an unpredictable way
 - Use of dyno roller data for TP2 leads to uncertainty and variation
- Internal validation and reality checks needed
 - How to verify supplied K factors, particularly for gearbox
 - Solving for K by setting $TP1 = TP2$ can provide a reality check
- Novel hybrid architectures require additional care
 - TP1 and TP2 are comparable for simple parallel hybrids
 - Other architectures where power flow follows separate paths to the destination will need additional K factors and instrumentation



Summary and conclusions

- US, Canada, and European Commission JRC have tested 11 hybrid electric vehicles and applied the draft procedure
 - Areas of ambiguity have been identified and clarified
 - Difference between TP1 and TP2 observed and sources identified
- The EVE IWG continues its work to develop the procedure
 - Minimize or eliminate the difference between TP1 and TP2
 - Continue to validate the evolving procedure by testing additional vehicles and novel hybrid configurations